

PVP 2009-77847  
Summary Review of Updated ASME  
PCC-1 20XX “Guideline For Pressure  
Boundary Bolted Flanged Joint Assembly”

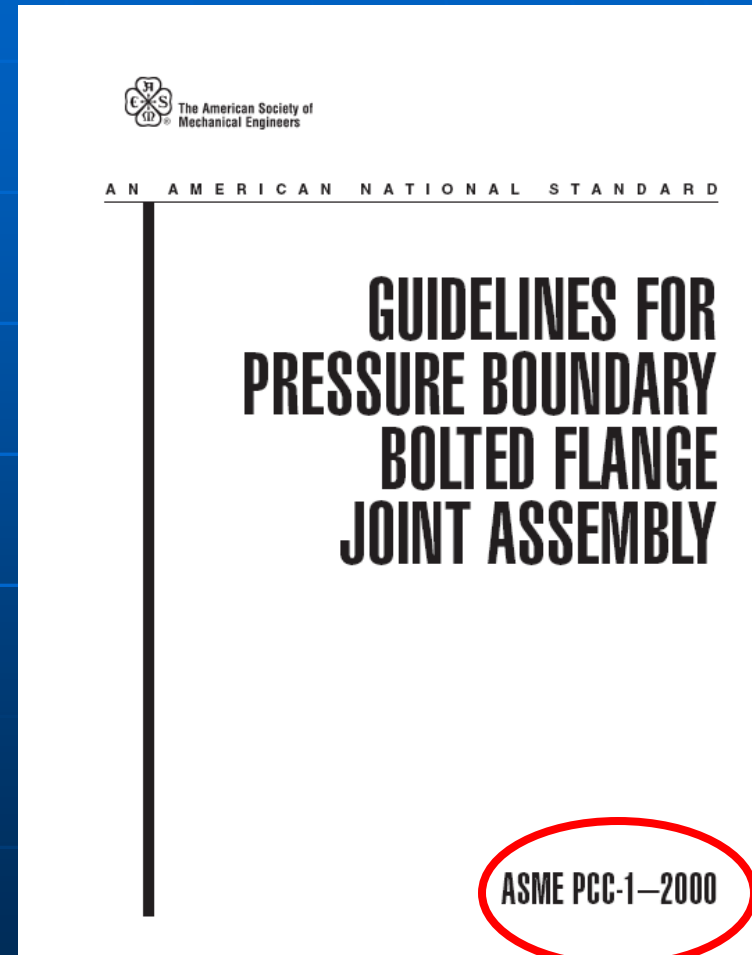
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Presented To:  
ASME PVP-2009  
Prague, Czech Republic  
July 28, 2009

# Background

- ASME Post Construction Committee  
PCC-1-2000 Update Project
- New Sub-Committee on Flange Joint Assembly
- Updates Include:
  - Expanded Guidance/Best Practices
  - Updated & New Appendices



# High Impact Appendices

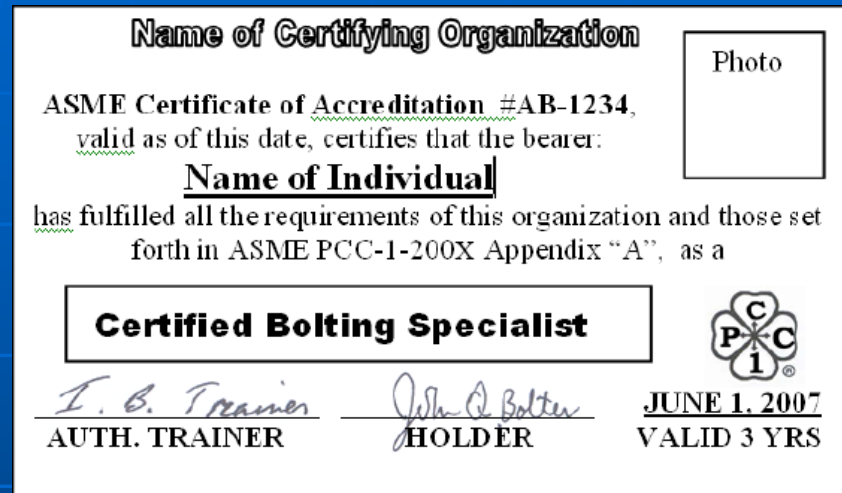
## ■ **Appendix A:**

### Training, Qualification and Certification of Joint Assembly Personnel

- “Guidelines for uniform criteria for training, qualifying and certifying bolted joint assembly personnel”
- Certification levels:
  - Certified Bolting Specialist
  - Certified Senior Bolting Specialist
  - Certified Bolting Specialist Instructors
- Requirements for Accreditation of Certifying Organizations

# Appendix A (Cont.)

- Credentials



- Classroom instruction
- Practical flange assembly examination
- Written examination

# Appendix A (Cont.)

- Bolting Specialist: Classroom Instruction/Training Fundamentals
  - General health and safety precautions
  - Equipment requirements
  - Principles of bolt elongation, bolt load and bolt stress
  - Functionality of gasket and seal
  - Gasket types and their limitations
  - Bolt types and their limitations
  - Identification of correct joint components
  - Manual torque wrench joint tightening
  - Importance of thread lubricants
  - Techniques for load control
  - Calibration and maintenance of bolt tightening equipment
  - Inspection and reporting of defects and faults
  - Preparing a joint for closure
  - Gasket handling, preparation, and installation
  - Safe joint dis-assembly and assembly
  - Correct use of additional joint components (washers, etc.)
  - Joint assembly QA procedures, certification and records
  - Joint dis-assembly

# Appendix A (Cont.)

- Bolting Specialist: Additional Training Endorsements
  - Piping specific training (Required)
  - Powered Equipment Endorsement
  - Heat Exchanger Endorsement
  - Special Joint Endorsements

# Appendix A (Cont.)

- Bolting Specialist: Practical Flange Assembly Demonstrations and Examination
  - Importance of gasket placement/alignment
  - Importance of joint alignment
  - Importance of bolt assembly patterns
  - Importance of bolt assembly pattern and correct gasket selection
  - Reaction of different gasket types to standard tightening procedure
  - Demonstration of effect of lubricants
  
- Re-Certification exam every three years

# High Impact Appendices

## ■ Appendix D:

Guidelines for Allowable Gasket Contact Surface Flatness and Defect Depth

**Table D-1 Flange Seating Face Flatness Tolerances (inches)**

	Hard Gaskets	Soft Gaskets
Acceptable Variation in Circumferential Flange Seating Surface Flatness	$T1 < 0.006$ <u><a href="#">in.</a></u>	$T1 < 0.01$ <u><a href="#">in.</a></u>
Acceptable Variation in Radial (across surface) Flange Seating Surface Flatness	$T2 < 0.006$ <u><a href="#">in.</a></u>	$T2 < 0.01$ <u><a href="#">in.</a></u>
Maximum Acceptable Pass-Partition surface height vs. flange face	$- 0.01$ <u><a href="#">in.</a></u> $< P < 0.0$ <u><a href="#">in.</a></u>	$- 0.02$ <u><a href="#">in.</a></u> $< P < 0.0$ <u><a href="#">in.</a></u>

# High Impact Appendices

- **Appendix D:**

Guidelines for Allowable Gasket Contact Surface Flatness and Defect Depth

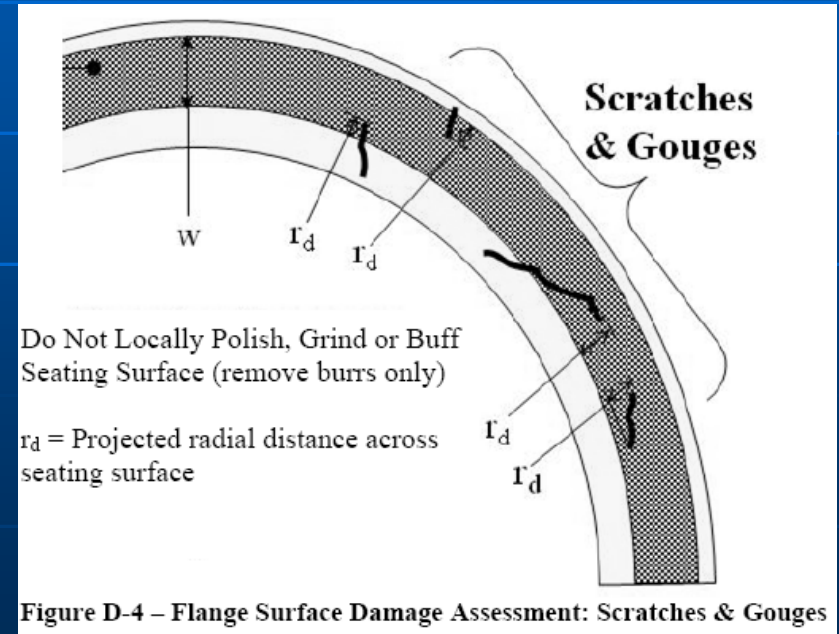
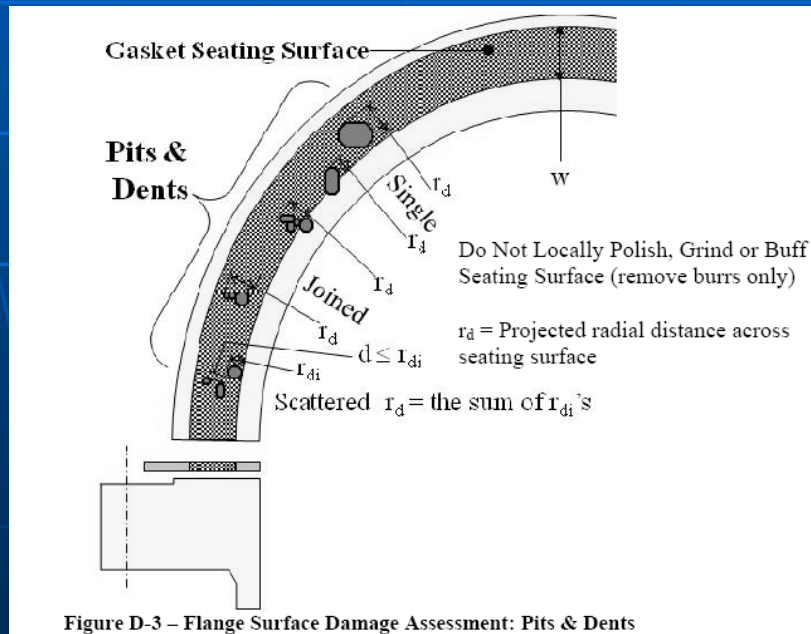
**Table D-2 Allowable Defect Depth vs. Width Across Face (inches)**

	Hard Faced Gaskets	Soft Faced Gaskets
$r_d < w/4$	$< 0.03$ <u><a href="#">in.</a></u>	$< 0.05$ <u><a href="#">in.</a></u>
$w/4 < r_d < w/2$	$< 0.01$ <u><a href="#">in.</a></u>	$< 0.03$ <u><a href="#">in.</a></u>
$w/2 < r_d < 3w/4$	Not Allowed	$< 0.005$ <u><a href="#">in.</a></u>
$r_d > 3w/4$	Not Allowed	Not Allowed

# High Impact Appendices

## ■ Appendix D:

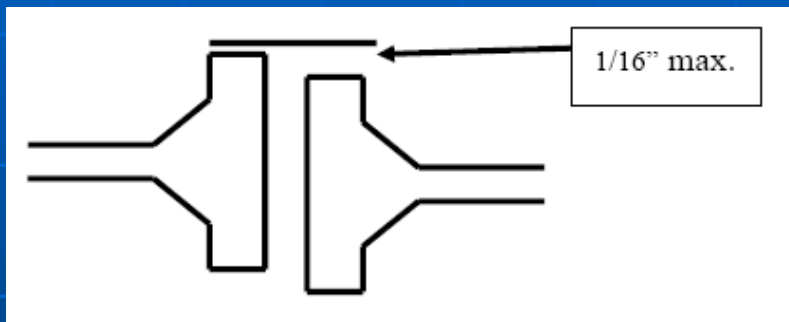
### Guidelines for Allowable Gasket Contact Surface Flatness and Defect Depth



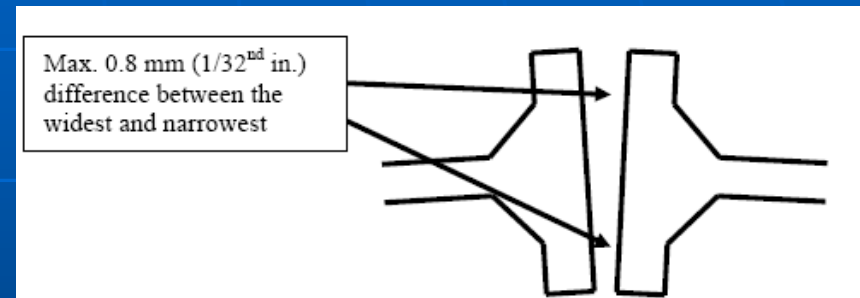
# High Impact Appendices

## ■ Appendix E:

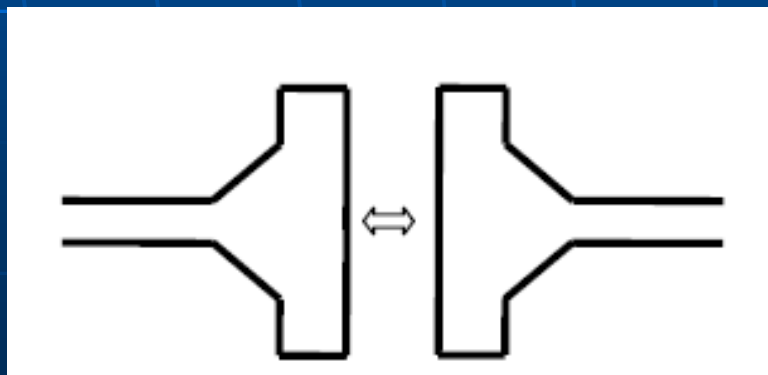
### Flange Joint Alignment Guidelines



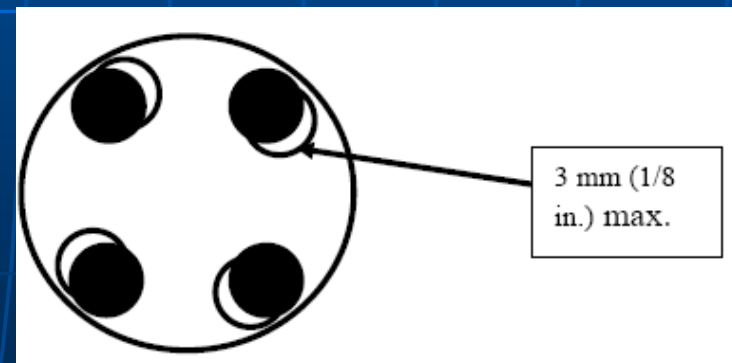
Centerline



Parallelism



Flange Face Gap



Rotational

# High Impact Appendices

- **Appendix F:**

Alternatives to Legacy Tightening Sequence/Pattern

1. Five alternative (non-Legacy) assembly procedures detailed
2. Guidelines for developing and qualifying additional alternative assembly procedures

# High Impact Appendices

## ■ Appendix F: Alternatives to Legacy Tightening Sequence/Pattern

### ALTERNATIVE ASSEMBLY PATTERN #1

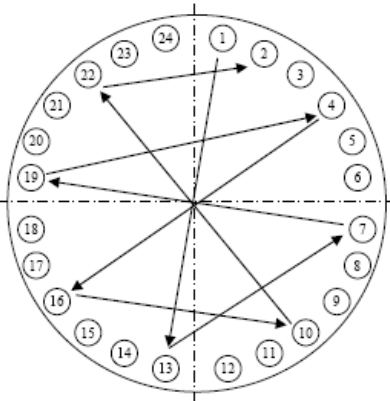
This pattern uses the same pattern as the Legacy method, however the stress levels are increased more rapidly, which allows fewer pattern passes to be performed and less overall effort. This method has been successfully applied in limited applications across the full range of gaskets and joint configurations.

Pass #1a: Proceed in the pattern outlined below and tighten the first four bolts at 20-30% of Target Torque

Pass #1b2: Tighten the next four bolts at 50-70% of final load

Pass #1c & 23: Tighten all subsequent bolts at 100% of final load until all pattern passes are complete

Pass #3 onward4: Tighten in circular passes until the nuts no longer turn



#### 24 Bolt Example:

Pass 1a – 20-30% of Target Torque:  
1,13,7,19

Pass 1b – 50-70% of Target Torque:  
4,16,10,22

Pass 1c - 100% of Target Torque:  
2,14,8,20 - 5,17,11,23 - 3,15,9,21 -  
6,18,12,24

Pass 2 (If 2<sup>nd</sup> pattern pass specified) -  
100% of Target Torque  
1,13,7,19 - 4,16,10,22 - 2,14,8,20 -  
5,17,11,23 - 3,15,9,21 - 6,18,12,24

Pass 3 & Subsequent onward - 100% of  
Target Torque, in rotational-circular  
pattern, until nuts do not turn.  
1,2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,  
17,18,19,20,21,22,23,24 - 1,2,3, etc....

Detailed Instructions

# High Impact Appendices

- **Appendix F:**  
 Alternatives to Legacy Tightening Sequence/Pattern

<b>Alternative Pattern #1</b>  <b>MODIFIED STAR LEGACY PATTERN</b>	The same as the Legacy <del>method</del> <u>pattern</u> , however the stress levels are increased more rapidly, allowing fewer pattern passes to be performed and less overall effort. This method has been successfully applied in limited applications across the full range of gaskets and joint configurations.	1 <sup>st</sup> Four to Six Bolts, <u>Star Pattern</u>	Next Four to Six Bolts, <u>Star Pattern</u>	Remaining Bolts, <u>Star Pattern</u>	All Bolts, <u>Star Pattern</u>	All Bolts, <u>CircularRet</u> <u>ational</u> <u>Pattern</u> Until No Further Nut Movement	For soft gaskets <sup>1</sup> , a minimum of 2 pattern passes are required. For hard gaskets <sup>2</sup> a minimum of one pattern pass is required. For problematic joints, it is recommended that an additional pattern pass be completed above the minimum required.
Percent Of Final Torque		30%	70%	100%	100%	100%	

Summary Instructions

# High Impact Appendices

- **Appendix F:**  
Alternatives to Legacy Tightening Sequence/Pattern

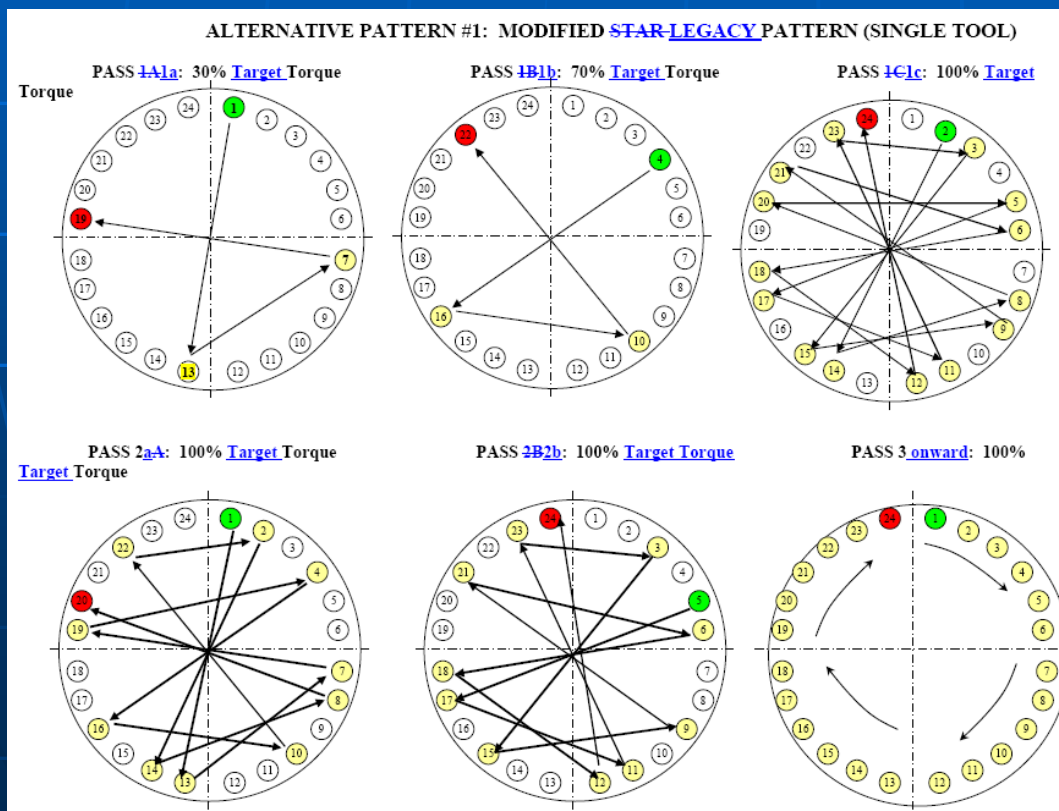


Diagram Examples

# High Impact Appendices

## ■ Appendix M: Washer Usage Guidance

Washer usage guidance

Material Type	Single-Use <sup>NOTE (1)</sup>	Reuse <sup>NOTE(2)</sup>
1	425 deg C (800 deg F)	205 deg C (400 deg F)
4	540 deg C (1000 deg F)	400 deg C (750 deg F)
5	650 deg C (1200 deg F)	425 deg C (800 deg F)
6	815 deg C (1500 deg F)	550 deg C (1025 deg F)

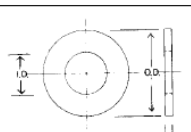
(1) Single-use service temperature limits are based on replacement whenever the existing washer has been exposed to temperature in excess of the corresponding reuse limit.  
(2) Reuse service temperature limits are based on metallurgical concerns (softening) for the washer material.

Purchase specification for through-hardened washers

Dimensional requirements

Dimensional tolerances

TABLE M-4 Dimensional Requirements for [Inch-U.S. Customary Washers](#) <sup>NOTE(1)</sup>



Nominal Size, (in.)	Outside Diameter (OD)		Inside Diameter (ID)		Thickness (T)	
	(mm)	(in.)	(mm)	(in.)	(mm)	(in.)
1/2	27.0	1.063	14.3	0.563	3.2	0.125
5/8	33.4	1.313	17.5	0.688	4.0	0.156
3/4	38.1	1.500	20.7	0.813	4.8	0.188
7/8	43.6	1.718	23.8	0.938	5.6	0.219
1	50.0	1.968	27.0	1.063	6.4	0.250
1 1/8	54.8	2.156	30.2	1.188	6.4	0.250
1 1/4	60.3	2.375	33.4	1.313	6.4	0.250
1 3/8	65.9	2.593	36.5	1.438	6.4	0.250

# High Impact Appendices

## ■ Appendix O: Assembly Bolt Stress Determination

Method for establishing assembly bolt stresses considering:

- Sufficient assembly and operating gasket stress
- Maximum gasket stress
- Maximum bolt stress
- Permanent flange deformation
  1. Simple Approach
  2. Joint Component Approach

# High Impact Appendices

## ■ Appendix O: Assembly Bolt Stress Determination

Example Analysis:

Table O-9\_ Example Bolt Stress and Torque Tables for Weldneck Flanges, SA-105 & SA193-B7 Material, Spiral Wound with inner ring.

Example Limits Used in the Analysis:

$Sb_{min} = 35$  ksi  
 $Sb_{max} = 75$  ksi  
 $Sb_{sel} = 30$  ksi  
 $Sg_{max} = 40$  ksi  
 $Sg_{min-S} = 12.5$  ksi  
 $Sg_{min-O} = 6$  ksi  
 $\theta_{g_{max}} = 1.0^\circ$   
 $Sf_{max}$  from Table O-3

	Min. Calculated Bolt Stress (ksi)					
	150	300	600	900	1500	2500
2	75	56	56	43	43	35
2.5	75	44	44	40	40	35
3	75	63	64	60	38	35
4	75	75	75	49	42	35
5	75	75	75	48	36	35
6	75	75	74	56	38	35
8	75	75	75	44	35	35
10	75	75	62	40	35	35
12	75	75	66	49	35	35
14	75	63	54	44	35	
16	75	63	58	42	35	
18	75	68	62	45	35	
20	75	74	57	38	35	
24	75	68	50	35	35	
26	37	37	41	35		
28	35	37	38	35		
30	37	42	41	35		
32	35	37	35	35		
34	35	42	36	35		
36	35	37	35	35		
38	37	75	39	35		
40	35	68	36	35		
42	37	71	35	35		
44	35	68	35	35		
46	37	69	35	35		
48	37	63	35	35		

- = Limited by Min. Bolt Stress
- = Limited by Max. Bolt Stress
- = Limited by Max. Gasket Stress
- = Limited by Max. Flange Stress

# High Impact Appendices

## ■ Appendix P: Guidance On Troubleshooting Flanged Joint Leakage Incidents

- Investigative and Diagnostic Evaluation Guide
- Checklist of Flange Design and Acceptable Practice Considerations
- Leakage Problems and Potential Solutions

LHT - Leak during hydro-test.

LIO - Leak during heat-up or initial operation.

LCU - Leak corresponding to thermal or pressure upset.

LTO - Leak after several months operation - piping.

LDS - Leak during shut down.

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Sample Flange Joint Leak Report

- Date: \_\_\_\_\_
- Unit: \_\_\_\_\_
- Equipment: \_\_\_\_\_
- Joint Identification: \_\_\_\_\_
- Is or Drawing#: \_\_\_\_\_
- Flange Size / Press. Class: \_\_\_\_\_
- Ordnance Material / Type: \_\_\_\_\_
- Describe the use of the joint, i.e. channel cover.
- Circle the best descriptive location and orientation.
- Flange Temp. / Bolt Temp.: \_\_\_\_\_
- Describe Leak Type: (wisp, droplet, stream, emission)
- Leak Timing: (during, @ 1<sup>st</sup> startup, @ later startups, @ cool down, @ \_\_\_\_\_ months operation, Other: \_\_\_\_\_)
- Bolt Lubricant Condition: \_\_\_\_\_

14. Mark the leak location.

15. Measure the gap between the flanges at 4 locations.

16. Measure the torque it takes to move the nuts. Record applied torque during tightening.

17. Mark nuts with the following marks after applying torque:

Nuts do not turn = 0	Nuts turn slightly = X
Nuts turn = XX	Nuts turn very easily = XXX

18. Leakage status: No Change \_\_\_ Reduced \_\_\_ Stopped \_\_\_

19. Advise conditions: \_\_\_\_\_

20. Comments: \_\_\_\_\_

21. Recommendations: \_\_\_\_\_

22. Name: \_\_\_\_\_

23. Signature: \_\_\_\_\_

## ***(LTO) Leak after long term (months) of operation***

Tell-Tale Signs	Possible Cause	Potential Solutions
Gasket structure/filler missing or no longer flexible or compliant	Gasket Chemical Degradation (chemical decomposition, <a href="#">oxidation</a> , etc.)	Change gasket type
Spring hangers incorrect, support lift-off, incorrectly placed restraints	Improper <a href="#">Pipe-pipe</a> support or restraint	See same item in LIO
Bolts are not tight on inspection	Bolt load loss due to long term gasket creep	See same item in LIO
Bolts not tight on inspection, Obvious <a href="#">Gasket-gasket</a> deterioration, gasket structure no longer sound	Physical gasket degradation, gasket unsuitable for operating temperature	Replace gasket with a type suitable for operating conditions
Gasket structure no longer sound (double jacket broken or windings buckled) Marks on gasket surface corresponding to radial flange face movement	Gasket Physical Degradation due to flange differential radial movement	<ul style="list-style-type: none"> <li>• Remove all flange face nubbins</li> <li>• Replace gasket with one capable of radial shear and taking greater abrasion(spiral wound, kammprofile or corrugated)</li> </ul>

# Publication Status

## PCC-1-2000 Update:

- Post Construction Standards Committee (PCSC) approval Of Updated PCC-1 (minus Appendix A)
- Appendix A recirculation ballot submitted to PCSC
  - Ballot closes July 28, 2009 (minor editorial comments rcvd)
  - 2<sup>nd</sup> recirculation ballot closes August 18, 2009
- Submit complete PCC-1 Update, including Appendix A, for review by Board of Pressure Technology Codes and Standards (BPTCS) ~August 19, 2009
- Submit to Codes & Standards Publishing ~August 19, 2009
- Publication expected late 2009